Loss Calculation and Thermal Analysis of Axial AMB in HTR-PM Helium Circulator

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\textbf{Abstract} — The helium circulator is one of the important components of the High Temperature Gas-cooled Reactor Pebble Bed Module (HTR-PM). It uses the electromagnetic suspension technology to support the rotor, the so-called magnetic bearing technology. Considering the performance and safety issues on the axial active magnetic bearing(AMB) caused by potential high temperature, this paper calculates copper loss, iron loss and wind loss in axial AMB and simulates temperature field under a normal operating condition, and compares the results with the experimental data.

\textbf{Index Terms} — Axial magnetic bearing, copper loss, helium circulator, iron loss, temperature field, wind loss.

\section*{I. INTRODUCTION}

The HTR-PM is one of the most promising candidates for the next generation reactors. It’s renowned for its inherent safety, system simplification and high power generation efficiency.

The helium circulator is the key equipment in the primary loop of the HTR-PM and installed at the output at the steam generator. It drives helium coolant with an average temperature of 750\degree C to circulate in the primary loop for heat exchange released by the nuclear reaction \cite{1}. Figure 1 shows a cross section of the HTR-PM reactor.

The rotor of helium circulator is supported by axial active magnetic bearings (AMB) instead of mechanical bearings for its excellent performance of non-contact, non-polluting and high-speed characteristics \cite{2}. But there still exist copper loss, iron loss and wind loss in magnetic bearings, leading to an increase in temperature and affecting the performance of the AMB system under some special conditions. Therefore, it is necessary to calculate the losses and the temperature field of the axial AMB in HTR-PM.

\begin{figure}[h]
\centering
\includegraphics[width=\textwidth]{Fig.1.png}
\caption{Cross section of the primary loop of the HTR-PM (1. reactor core; 2. side reflector and carbon thermal shield; 3. core barrel; 4. reactor pressure vessel; 5. steam generator; 6. steam generator vessel; 7. coaxial gas duct; 8. water-cooling panel; 9. blower; 10. fuel discharging tube).}
\end{figure}
Researches have been done including theoretical and experimental studies on iron loss of laminated magnets: Karsarda proposed the calculation formulas for eddy current and hysteresis loss of laminated magnets [3-4]; Marinescu [5] discussed the possibility of calculating the eddy current loss of permanent magnet bearings in two dimensions. For solid axial bearings with a single-coil structure, Sun [6-7] proposed analytical solutions and finite element solutions for eddy current losses considering electromagnet temperature coupling.

As for the research of AMB temperature field, Sun [6-7] solved the temperature field while neglecting the wind loss and considering eddy current losses as an even distributed internal heat source in the mathematical model of the temperature field.

In this paper, based on the actual operation current data of the HTR-PM axial AMB, the losses of the axial bearing are studied. Furthermore, the temperature field is modeled and theoretically analyzed on ANSYS platform. The theoretical simulation analysis results are compared with the actual temperature data. The simulation results are in accordance with the actual operating data, which provides a theoretical model support and an effective calculation method for the subsequent optimization design of the axial AMB.

II. GEOMETRIC MODEL OF AXIAL AMB IN HTR-PM

The HTR-PM axial bearing is used to support the gravity of the rotor in the helium circulator. The rotor has a length of 3.3 meters and a mass of 4000 kg. The speed at normal operation is 4000 r/min. The structure of axial magnetic bearing is shown in Fig. 2, including 4 main sections: thrust disk, stators, coils and rotor. The parameter is shown in Table 1.

![Fig. 2. Structure of the axial AMB (1. upper stator inner coil; 2. upper stator outer coil; 3. lower stator inner coil; 4. lower stator outer coil; 5. upper stator; 6. lower stator; 7. thrust disk).](image)

<table>
<thead>
<tr>
<th>Parameter</th>
<th>Unit</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Gap between stator and rotor</td>
<td>mm</td>
<td>1</td>
</tr>
<tr>
<td>Inner magnetic pole area, $A_i$</td>
<td>mm²</td>
<td>7868.8</td>
</tr>
<tr>
<td>Central magnetic pole area, $A_c$</td>
<td>mm²</td>
<td>15009.2</td>
</tr>
<tr>
<td>Outer magnetic pole area, $A_o$</td>
<td>mm²</td>
<td>9982.1</td>
</tr>
<tr>
<td>Trust disk diameter</td>
<td>mm</td>
<td>300</td>
</tr>
<tr>
<td>Vacuum permeability, $\mu_0$</td>
<td>N/A²</td>
<td>$4\pi \times 10^{-7} \approx 1.256637 \times 10^{-6}$</td>
</tr>
<tr>
<td>Winding turns</td>
<td>-</td>
<td>78, 78</td>
</tr>
</tbody>
</table>

The actual HTR-PM axial bearing structure is complex. In order to save computing resources, the geometric model used in this paper simplifies the connecting nuts between the components, the chamfers, and the disassembly tooling of the thrust disk, and the windings are processed solidly. The model is shown in Fig. 3 below. The following three assumptions are the bases for the loss calculation and temperature field analysis:

1. The eddy current has the same effect on each strand of the axial AMB, and the loss is evenly distributed on the windings;
2. The temperature difference inside the bearing is not large and the radiation is neglected;
3. The actual solid surface is ideally flat and the thermal contact resistance is ignored.

![Fig. 3. Geometric model of axial AMB.](image)

III. LOSSES CALCULATION

While the axial AMB realizes the stable support of the rotor in HTR-PM, the winding needs a large current to generate the electromagnetic force that balances the gravity and aerodynamic force of the rotor, and there will also be current fluctuations in the actual operation. During
high-speed rotating, there will be copper loss and iron loss in the stator and thrust disk, and wind loss on the surface of the thrust disk. Based on the actual operating condition and current data, the copper loss, iron loss, and wind loss of the axial AMB are analyzed.

A. Operating condition

Excluding start, shutdown and some special abnormal conditions, the HTR-PM helium circulator are usually in normal operating conditions, the following analysis and calculation are based on this condition. The operating parameters of the HTR-PM helium circulator are shown in Table 2.

Table 2: Main operating parameters of helium circulator

<table>
<thead>
<tr>
<th>Parameter</th>
<th>Unit</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Medium</td>
<td>-</td>
<td>Helium</td>
</tr>
<tr>
<td>Rated speed</td>
<td>r/min</td>
<td>4000</td>
</tr>
<tr>
<td>Mass flow</td>
<td>kg/s</td>
<td>96</td>
</tr>
<tr>
<td>Inlet pressure</td>
<td>Mpa</td>
<td>7</td>
</tr>
<tr>
<td>Speed range</td>
<td>%</td>
<td>20~105</td>
</tr>
<tr>
<td>Medium density</td>
<td>kg/m³</td>
<td>6.33</td>
</tr>
<tr>
<td>Circulator chamber ambient temp</td>
<td>°C</td>
<td>65</td>
</tr>
</tbody>
</table>

B. Copper loss

The gravitational force and aerodynamic force of the rotor are balanced by the electromagnetic force generated by the winding’s current. The loss caused by the current passing through the winding is called the copper loss. The actual electrical parameters of the upper and lower coils are measured at 25°C and the results are shown in Table 3, and the copper loss of the axial bearings can be calculated by formula (1):

$$P_{Cu} = Ri^2.$$  \hspace{1cm} (1)

Among them, $P_{Cu}$ is the copper loss of axial bearings in W, $R$ is the resistance of the winding in Ω, $i$ is the current in A.

Table 3: Resistance of axial AMB

<table>
<thead>
<tr>
<th>Parameters</th>
<th>Unit</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Upper winding inner coil resistance</td>
<td>Ω</td>
<td>0.38</td>
</tr>
<tr>
<td>Upper winding outer coil resistance</td>
<td>Ω</td>
<td>0.37</td>
</tr>
<tr>
<td>Lower winding inner coil resistance</td>
<td>Ω</td>
<td>0.38</td>
</tr>
<tr>
<td>Lower winding outer coil resistance</td>
<td>Ω</td>
<td>0.37</td>
</tr>
</tbody>
</table>

The measured average value of the upper and lower winding currents under normal operation is 36.5A and 23.7A respectively. Through the calculation of formula (1), the copper loss during normal operation are shown in Table 4.

Table 4: Copper losses of axial AMB

<table>
<thead>
<tr>
<th>Parameters</th>
<th>Unit</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Upper stator inner coil copper loss</td>
<td>W</td>
<td>506</td>
</tr>
<tr>
<td>Upper stator outer coil copper loss</td>
<td>W</td>
<td>492</td>
</tr>
<tr>
<td>Lower stator inner coil copper loss</td>
<td>W</td>
<td>213</td>
</tr>
<tr>
<td>Lower stator outer coil copper loss</td>
<td>W</td>
<td>207</td>
</tr>
</tbody>
</table>

C. Iron loss

The classic theory of iron loss point that iron loss can be divided into eddy current loss, hysteresis loss and other losses. Eddy current loss is the energy loss caused by the induced current generated in the conductor when it moves in a non-uniform magnetic field or is in a time-varying magnetic field. Hysteresis loss refers to the energy consumed by the hysteresis of the ferromagnetic magnet during repetitive magnetization.

Iron loss mainly exists in the stator and thrust disks of the axial AMB, which material is 40CrNiMoA soft magnetic material. Because of manufacturing processes and other reasons, axial AMB are made in solid structure, for which means the eddy current loss is relatively large compared with the hysteresis loss. Therefore, only the eddy current losses are considered in the calculation of iron loss, which can be described by equation (2):

$$P_{iron} = P_{eddy}(x, y, z).$$ \hspace{1cm} (2)

Among them, $P_{iron}$ is the iron loss of axial bearings in W, $P_{eddy}$ is the eddy current loss of axial bearings in W.

In order to accurately calculate the three-dimensional distribution of the eddy current loss in the solid double-coil axial AMB, the actual speed and the current data are obtained from actual parameters under normal operating conditions as the bases of calculation.

The Fourier-transformation of the upper and lower axial bearing current data in the running state is performed, and the main frequency is extracted. The results are shown in Fig. 4 and Fig. 5, which show the actual current of the axial bearing has a significant dominant frequency of 56 Hz.

![Fig. 4. Fourier Transform of upper axial AMB current.](image-url)
Fig. 5. Fourier Transform of lower axial AMB current.

The fitting current function and the comparison between fitting current and the actual current are shown in Table 5, Fig. 6 and Fig. 7.

Table 5: Fitted currents
<table>
<thead>
<tr>
<th>Current</th>
<th>Fitting Function</th>
</tr>
</thead>
<tbody>
<tr>
<td>Upper axial AMB current</td>
<td>$I = 36.64 + 8.18\cos 352.2t - 18.55\sin 352.2t$</td>
</tr>
<tr>
<td>Lower axial AMB current</td>
<td>$I = 23.74 - 5.37\cos 353.4t - 10.13\sin 353.4t$</td>
</tr>
</tbody>
</table>

Fig. 6. Upper fitted current curve and original current.

Fig. 7. Lower fitted current curve and original current.

The eddy current loss distribution of the stator and thrust disk is obtained by the finite element method. The results are shown in Fig. 8 to Fig. 10, the eddy current loss is concentrated around the windings in the stator and concentrated on the corresponding positions of the windings on the thrust disk.

Table 6: Eddy current loss calculation
<table>
<thead>
<tr>
<th>Part</th>
<th>Unit</th>
<th>Eddy Current Loss</th>
</tr>
</thead>
<tbody>
<tr>
<td>Upper stator</td>
<td>W</td>
<td>2622</td>
</tr>
<tr>
<td>Lower stator</td>
<td>W</td>
<td>787</td>
</tr>
<tr>
<td>Thrust disk</td>
<td>W</td>
<td>559</td>
</tr>
</tbody>
</table>

D. Wind loss

As shown in Fig. 1, the high-speed rotating thrust disk is supported by the axial AMB, so there is a forced flow of gas in the air gap. The friction generated between the gas and the rotor is wind loss. Wind losses mainly exist on the disk surfaces and end surfaces of high-speed rotating thrust disks. The wind loss is not related to the electrical parameters of the axial AMB, the calculation
of the wind loss can refer to the research of the rotating machine.

End-face wind loss can be based on the empirical formula of Von Karman’s plate turbulence model as formula (3):

$$P_w = 0.311 \left( \frac{s}{r} \right)^{2/4} \mu^{1/4} \rho^{1/4} \omega^{1/4} r^{9/2}.$$  (4)

Among them, $s$ is the air gap width in m, $r$ is the radius of the disk in m, and $\mu$ is the dynamic viscosity of the gas in Pa·s. The calculation is measured on one side.

The parameter for wind loss calculation and results are shown in Table 7 and Table 8.

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Table 7: Parameter for wind loss calculation

<table>
<thead>
<tr>
<th>Parameters</th>
<th>Unit</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Ambient helium</td>
<td>°C</td>
<td>65</td>
</tr>
<tr>
<td>temperature</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Helium pressure</td>
<td>MPa</td>
<td>7</td>
</tr>
<tr>
<td>Helium density</td>
<td>kg/m³</td>
<td>9.686420455</td>
</tr>
<tr>
<td>Helium dynamic</td>
<td>Pa·s</td>
<td>2.1592×10⁻⁵</td>
</tr>
<tr>
<td>viscosity</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Helium kinematic</td>
<td>m²/s</td>
<td>2.2291×10⁻⁶</td>
</tr>
<tr>
<td>viscosity</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Radium of thrust disk,</td>
<td>mm</td>
<td>300</td>
</tr>
<tr>
<td>$r$</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Length of thrust disk,</td>
<td>mm</td>
<td>32</td>
</tr>
<tr>
<td>$L$</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Air gap width</td>
<td>mm</td>
<td>0.8</td>
</tr>
<tr>
<td>Rotational angular</td>
<td>rad/s</td>
<td>418.67</td>
</tr>
<tr>
<td>velocity</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

Table 8: Wind loss results

<table>
<thead>
<tr>
<th>Parameters</th>
<th>Unit</th>
<th>Value</th>
</tr>
</thead>
<tbody>
<tr>
<td>Wind loss on end</td>
<td>W</td>
<td>1673</td>
</tr>
<tr>
<td>surface</td>
<td></td>
<td></td>
</tr>
<tr>
<td>Wind loss on one</td>
<td>W</td>
<td>2089</td>
</tr>
<tr>
<td>disk surface</td>
<td></td>
<td></td>
</tr>
</tbody>
</table>

E. Results analysis

Through calculation and analysis of the copper loss, iron loss and wind loss of the HTR-PM helium circulator axial AMB under normal operating conditions, the results are shown in Table 9. The total loss in the axial bearing is 9446 W, among which the wind loss accounted for the largest proportion, followed by the proportion of iron loss, and the copper loss accounted for the smallest proportion. The results are in good agreement with the actual project conditions.

Table 9: Percentage of each loss

<table>
<thead>
<tr>
<th>Loss</th>
<th>Unit</th>
<th>Value</th>
<th>Percentage</th>
</tr>
</thead>
<tbody>
<tr>
<td>Iron loss</td>
<td>W</td>
<td>3968</td>
<td>42.0%</td>
</tr>
<tr>
<td>Copper loss</td>
<td>W</td>
<td>1418</td>
<td>15.0%</td>
</tr>
<tr>
<td>Wind loss</td>
<td>W</td>
<td>4060</td>
<td>43.0%</td>
</tr>
<tr>
<td>Total loss</td>
<td>W</td>
<td>9446</td>
<td>100%</td>
</tr>
</tbody>
</table>

IV. TEMPERATURE FIELD ANALYSIS AND CALCULATION

Based on the results of the losses of the axial bearings in the HTR-PM helium circulator, the mathematical model of temperature field is established for the analysis.

A. Mathematical model of temperature field

According to the law of Fourier heat conduction, the three-dimensional steady-state temperature field of an axial AMB can be described as a differential conduction equation in Cartesian coordinates, as shown in equation (5):

$$\rho C_p \frac{\partial T}{\partial t} = \lambda \left( \frac{\partial^2 T}{\partial x^2} + \frac{\partial^2 T}{\partial y^2} + \frac{\partial^2 T}{\partial z^2} \right) + p_v.$$  (5)

Among them, $\rho$ is the density in kg/m³, $C_p$ is the constant-pressure specific heat in J/(kg·K), $\lambda$ is the thermal conductivity in W/(m·K), $p_v$ is the inner heat source in W/m³.

The eddy current loss is unevenly distributed, and the copper loss is evenly distributed in the conductors formed by each strand:

$$p_v = p_{eddy}(x,y,z) + p_{copper}.$$  (6)

The boundary conditions for the mathematical model of the temperature field include fixed heat flux and convection heat transfer boundaries:

$$q_n = \text{const},$$  (7)

$$-\lambda \frac{\partial T}{\partial n} = h(T_\text{wall} - T_f).$$  (8)

Among them, $q_n$ is the fixed heat flux, $h$ is the convective heat transfer coefficient.

The fixed heat flux $q_n$ is equal to the wind loss $q_w$ caused by the rotation in formula (3) and (4), and the convection heat transfer coefficient in formula (8) is calculated by the corresponding correlation [9].

B. Results and analysis

The temperature field results shown in Figs. 11 to 13 are obtained by solving equation (1)-(8) by the ANSYS under normal operating condition.

The temperature of the upper axial AMB is shown in the Fig. 11: axially, the temperature of the upper axial bearing increases from the surface close to the thrust disk to the surface far away from the thrust disk, because the high speed rotating causes a large forced convective heat...
transfer coefficient; radially, there are relatively high temperature areas at the tooth between the two slots and outer coil, because copper loss and eddy current loss are concentrated in this location. The maximum temperature of the upper axial bearing is 75.42°C.

![Temperature field of upper Axial AMB.](image1)

Fig. 11. Temperature field of upper Axial AMB.

The temperature field of the lower axial AMB is shown in the Fig. 12: the lower axial bearing have a similar temperature gradient distribution with the upper axial bearing. Because the lower axial bearing generates less copper loss and iron loss compared to the upper one, the temperature is slightly lower and the maximum temperature of the lower axial bearing is 71.56°C.

![Temperature field of lower Axial AMB.](image2)

Fig. 12. Temperature field of lower Axial AMB.

The temperature field of the thrust disk is shown in the Fig. 13. The average temperature of the upper surface of the thrust plate is higher than the lower surface, and due to the distribution of iron loss on the surface of the thrust plate, there are obvious high temperature areas on the upper and lower surfaces of the thrust disk. The high temperature areas are mainly concentrated on the thrust disk surface facing the coils on the stator, which is about 2°C higher than the average temperature on the surface. The maximum temperature of the thrust disk is 75.46°C.

![Temperature field of thrust disk.](image3)

Fig. 13. Temperature field of thrust disk.

Four test points are arranged near the inner and outer coils of the upper and lower axial AMB of the helium circulator, and the speed of the rotor during experiment is 3710r/min.

The test point temperature under the test condition and the maximum temperature in the calculation are shown in Table 10, both are below the allowable temperature limitation 150°C.

<table>
<thead>
<tr>
<th>Measuring Point</th>
<th>Experiment Results/°C</th>
<th>Calculation Results/°C (Maximum)</th>
<th>Relative Error/%</th>
</tr>
</thead>
<tbody>
<tr>
<td>Upper axial AMB inner coil temperature</td>
<td>85.0</td>
<td>75.4</td>
<td>-11.08%</td>
</tr>
<tr>
<td>Upper axial AMB outer coil temperature</td>
<td>82.6</td>
<td>74.0</td>
<td>-10.41%</td>
</tr>
</tbody>
</table>

There is certain error between the experimental results and the calculation results. The main reasons for
the are as follows:

(1) Windings are affected by factors such as the outsourcing insulating varnish, and the actual windings are different from solid copper which assumed in this article.

(2) When considering the copper loss, the effect of temperature increase on the resistance is ignored.

(3) The corresponding convective heat transfer coefficient at a specific location is inconsistent with the actual situation. Therefore, the calculation results will deviate from the actual situation.

V. CONCLUSION

(1) The article calculates the loss of the axial AMB of the HTR-PM helium circulator under the normal operating condition. The results show that there are concentrated areas of eddy current loss on the stator and the thrust disk, and the wind loss of the axial AMB can’t be ignored.

(2) In this paper, ANSYS is used to calculate the steady-state temperature field of the axial AMB in HTR-PM helium circulator under normal operation, the results show that the temperature is below the allowable temperature limitation. Comparing with the results of field tests, the calculated results are basically consistent with the experimental results, which show the correctness of the model and method, and thus can provide a reliable basis for the design of an axial AMB in helium circulation.

(3) From the simulation results, it can be seen that the upper stator temperature is higher than the lower stator temperature, the temperature is higher in the area where iron loss concentrates inside the axial bearing stator. There is also a higher temperature area where the iron loss concentrates on the surface of the thrust disk.

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REFERENCES


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